

This Page Is Inserted by IFW Operations
and is not a part of the Official Record

BEST AVAILABLE IMAGES

Defective images within this document are accurate representations of the original documents submitted by the applicant.

Defects in the images may include (but are not limited to):

- BLACK BORDERS
- TEXT CUT OFF AT TOP, BOTTOM OR SIDES
- FADED TEXT
- ILLEGIBLE TEXT
- SKEWED/SLANTED IMAGES
- COLORED PHOTOS
- BLACK OR VERY BLACK AND WHITE DARK PHOTOS
- GRAY SCALE DOCUMENTS

IMAGES ARE BEST AVAILABLE COPY.

**As rescanning documents *will not* correct images,
please do not report the images to the
Image Problem Mailbox.**

Friction Assembly And Method for Manufacturing Same

Field of the invention

5 This invention relates to a brake system, and more particularly to a friction assembly having a brake pad and a noise damping shim suitably used in a brake system on a vehicle and a method for manufacturing the same.

Background of the Invention

10 A typical brake system uses one or more friction assemblies. A friction assembly is a replaceable element. A friction assembly comprises a brake pad having a friction material mounted on a backing plate. For example, in a typical disc brake, a rotating disc or rotor rotates with a wheel of a vehicle. A pair of opposing friction assemblies are positioned on opposite sides of the rotor such that both friction materials face to the rotor. Outside the friction assemblies are a piston and a
15 corresponding caliper, respectively. When the brake is actuated and the piston and caliper are advanced closer to each other, the friction assemblies are pressed against the rotor between the piston and caliper. Thus, the friction materials contact the rotor to provide a stopping frictional force.

Significant forces are involved in applying the friction materials to the rotor.
20 Due to the relative movement during the engagement of the friction materials and the rotor, squeal noise can be produced. In order to reduce the squeal noise, it is known to attach a noise damping shim to the brake pad.

The noise damping shim is typically attached to the backing plate of the brake pad using pins or staples through holes formed on the backing plate and shim. In
25 this construction, the shim is subject to significant torques between the friction assembly and the piston or caliper pushing the shim against the brake pad during the braking operation. In order to minimize movements of the shim relative to the brake pad, the pins and holes need to be precisely dimensioned. Also, the use of pins increases the number of components to be assembled, which increases
30 manufacturing costs.

In some friction assemblies, a noise damping shim has tabs or hooks projecting outwardly from the peripheral of the shim. When the shim is attached to a backing plate, the hooks are bent and engaged to the backing plate. This type of friction assemblies are advantageous for requiring less number of components to be assembled compared to the friction assemblies that use pins. However, the provision of the tabs require additional shim materials and the hook bending process during the assembling requires a complicated assembling machine.

It is therefore desirable to provide a backing plate having improved engagement between a brake pad and a noise damping shim without involving undue extra manufacturing costs.

Summary of the invention

The present invention uses a coupling mechanism on a backing plate and a shim to facilitate efficient engagement of the backing plate and shim.

In accordance with an aspect of the invention, there is provided a friction assembly for a brake. The friction assembly comprises a brake pad and a noise damping shim. The brake pad has a friction material for producing friction when the brake is in use, and a backing plate having a friction supporting surface for supporting the friction material and a back surface opposite to the friction supporting surface, the back surface having a first coupler member. The noise damping shim has a first surface for engaging with the back surface of the backing plate, and a second surface for receiving a compression force when the brake is in use, the first surface having a second coupler member for engaging with the first coupler member provided on the back surface of the backing plate so as to couple the noise damping shim with the backing plate.

In accordance with another aspect of the invention, there is provided a backing plate for a friction assembly for a brake. The friction assembly has a brake pad including the backing plate and a friction material. The backing plate comprises a friction supporting surface for supporting a friction material for producing friction when the brake is in use, a back surface opposite to the friction supporting surface, the

back surface being adapted to receive a noise damping shim, and a first coupler member formed on the back surface for engaging with a second coupler member formed on the noise damping shim to couple the noise damping shim with the backing plate.

5 In accordance with another aspect of the invention, there is provided a method of assembling a noise damping shim with a brake pad having a backing plate and a friction material mounted on the backing plate. The method comprises the steps of providing a first coupler member on a back surface a friction assembly having a friction pad for producing friction when the brake is in use; and a backing plate having
10 a friction supporting surface for supporting the friction pad and a back surface opposite to the friction supporting surface; providing a second coupler member on a noise damping shim having a first surface adapted for engagement with the back surface of the backing plate, and a second surface adapted for receiving a compression force when the brake is in use; and coupling the second coupler
15 member provided on the noise damping shim with the first coupler member provided on the back surface of the backing plate so as to couple the shim with the backing plate.

Brief Description of the Drawings

20 For a better understanding of the present invention, and to show more clearly how it may be carried into effect, reference will now be made by way of example to the accompanying drawings. The drawings show preferred embodiments of the present invention, in which:

25 Figure 1 is a perspective explored view of a brake system in which a friction assembly in accordance with an embodiment of the present invention is used;

 Figure 2 is a perspective explored view of the friction assembly of Figure 1;

 Figure 3 is a perspective view of the friction assembly as assembled;

 Figure 4 is a partial cross-sectional view showing a coupler prior to engagement according to the embodiment;

Figure 5 is a partial cross-sectional view showing the coupler shown in Figure 4 after the engagement;

Figure 6 is a partial cross-sectional view showing another coupler prior to engagement according to another embodiment;

5 Figure 6a is a partial enlarged cross-sectional view showing details of the coupler shown in Figure 6; and

Figure 7 is a partial cross-sectional view showing the coupler shown in Figure 6 after the engagement.

10 Detailed Description of the Preferred Embodiments

Referring to Figure 1, a brake system 10 in which friction assemblies 20 according to an embodiment of the present invention is suitably used. This embodiment is described using a disc brake system for a vehicle as an example, however, the present invention may be applied to different types of brake systems.

15 The brake system 10 includes a pair of friction assemblies 20 arranged in opposite sides on a rotating disc or rotor 30, and brake actuating mechanism 40 provided outside the pair of friction assemblies 20.

20 The rotor 30 rotates with a wheel of the vehicle along a rotation axis 32. The rotor 30 has opposing flat end surfaces 34 extending parallel to each other and perpendicular to the rotation axis 32.

25 The brake actuating mechanism 40 includes a piston 42 and a corresponding caliper 44, each provided at outside of each friction assembly 20, respectively. When the brake system 10 is actuated, the piston 42 and caliper 44 are brought closer to each other by e.g., a hydraulic pressure mechanism. The piston 42 and caliper 44 press the friction assemblies 20 against the flat end surfaces 34 of rotor 30 from the opposing sides, thus a braking force is produced between the friction assembly 20 and the rotor 30.

Referring further to Figures 2-5, the friction assembly 20 is described in details.

Each friction assembly 20 has a brake pad 22 which comprises a friction material 24 and a backing plate 26. The friction material 24 is mounted on a front surface 26a of backing plate 26. The friction material 24 is typically mounted on the front surface 26a of backing plate 26 by moulding it on the front surface 26a with or without an adhesive layer between them. Different methods of mounting the friction material 24 on the front surface 26a of the backing plate 26 may be used. The backing plate 26 may have some holes 27 for manufacturing processes or other purposes.

Each friction assembly 20 also has a noise damping shim 28. As shown in Figures 2 and 3, the noise damping shim 28 is attached to a back surface 26b of the backing plate 26 using couplers 50. Figures 2 and 3 show two couplers 50 used for a single friction assembly 20. The two couplers 50 are provided at locations relatively close to two upper corners where the couplers 50 do not interfere with other holes 27 formed on the backing plate 26. The number and locations of couplers used in a single friction assembly may be determined depending on the desired design and application of the friction assembly.

Each coupler 50 comprises a first coupler element 60 formed on the back surface 26b of the backing plate 26, and a second coupler element 70 formed on the front surface 28a of the noise damping shim 28.

The first coupler element 60 has a stem 62 and a recess 64 surrounding the stem 62, as best seen in Figures 2 and 4. The recess 64 is defined on the back surface 26b inwardly into the backing plate 26 from an opening mouth 64b to a bottom 64a. The opening mouth 64b has generally a round shape. The diameter of the opening diminishes from mouth 64b to the bottom 64a. The recess 64 is deep enough to function as a coupler but shallow enough so that it does not affect to the necessary strength of the backing plate 26. The stem 62 generally has a cylindrical shape. Its connection end 62a is connected to the backing plate 26 at the bottom 64b of the recess 64. The head 62b of the stem 62 may project slightly outwardly over the back surface 26b of the backing plate 26 as shown in Figure 4, or it may remain within the recess 64, depending on application.

The second coupler element 70 has a rim 72 defining a hole 74. The rim 72 protrudes towards the backing plate 26 from a connection section 72a where the rim 72 is connected to the main body of the shim 28. The protruding rim body forms a tubular section 72b. The tubular section 72b has generally a cylindrical tubular shape with generally a constant inner diameter d_3 . The length of the tubular section 72b is shorter than the height of the stem 62 of the first coupler member 60. The connection section 72a is smoothly curved to provide an inner diameter d_4 which is larger than the diameter d_3 of the tubular section 72b. The connection section 72a does not have to be curved. For example, the connection section 72a may have one or more stepping shoulders to provide a larger diameter d_4 in different embodiments.

The shapes and dimensions of the stem 62 and recess 64 of the first coupler member 60 correspond to those of the rim 72 of the second coupler member 70 such that the stem 62 is received in the rim 72 and the rim 72 is received in the recess 64 when the first and second coupler members 60 and 70 are engaged.

The height h_1 of the stem 62 of the first coupler member 60 is typically between about 0.07 to 0.09 inches (about 2.24 to 3.98 mm) for backing plate 26 having a thickness t_1 of about 0.125 to 0.27 inches (about 3.18 to 6.86 mm). The diameter d_1 of the stem 62 is typically about 0.095 to 0.105 inches (about 2.41 to 2.67 mm). The diameter d_2 of the opening mouth 64b of the recess 64 is typically about 0.27 to 0.30 inches (about 6.85 to 7.62 mm). The height h_2 of the tubular section 72b of the rim 72 of the second coupler member 70 is typically between about 0.04 to 0.07 inches (about 1.02 to 1.78 mm) for noise damping shim 28 having a thickness t_2 of about 0.02 to 0.035 inches (about 0.51 to 0.89 mm). The inner diameter d_3 of the tubular section 72b is typically about 0.107 to 0.112 inches (about 2.72 to 2.84 mm). The inner diameter of the tubular section 72a is typically slightly larger than the diameter d_1 of the stem 62. The diameter d_4 of the curved section 72a is typically about 0.20 to 0.21 inches (about 5.08 to 5.33 mm). However, those dimensions may vary depending on application.

The first and second coupler elements 60 and 70 are provided at locations such that the noise damping shim 28 can be engaged with the backing plate 26 at a

desired location when the first and second coupler elements 60 and 70 mate each other.

It is preferable that the first and second coupler members 60 and 70 are integrally formed with the backing plate 26 and shim 28, respectively, as exemplified in Figure 4. For example, the first coupler element 60 may be formed on the backing plate 26 by stamping the backing plate material with suitable progressive dies or fine blanking dies to form stem 62 and recess 64.. Similarly, the second coupler member 70 may be formed by stamping the shim material using dies.

When the friction assembly 20 is assembled, the shim 28 is brought into contact with the backing plate 26. The rim 72 of the second coupler member 70 is received in the recession 64 and the stem 62 is received in the hole 74 defined by the rim 72. As the length of the tubular section 72b is shorter than the height of the stem 62 of the first coupler member 60, the head 62b of the stem 62 reaches the connection section 72a of the rim 72, as best seen in Figure 3.

In order to secure the engagement between the first coupler element 60 and the second coupler element 70, the head 62b of the stem 62 is pushed back towards the backing plate 26, as shown in Figure 5. This pushing force deforms the head 62b to close at least partially the opening 74 at the connection section 72a of the rim 72. As the diameter d_4 at the connection section 72a is larger than the diameter d_3 of the tubular section 72b, the deformed head 62b would not disengage through the tubular section 72b. Thus, the deformed stem 62 acts as a rivet, securing the engagement between the first coupler element 60 and the second coupler element 70.

Thus, the shim 28 can be efficiently assembled with the backing plate 26 without a need for extra components, such as a pin or staple.

In the above embodiment, the coupler 50 is coupled as a rivet. In a different embodiment, the coupler may be coupled by a snap-in fashion.

Figures 6, 6A and 7 show an example of a snap-in coupler 80. The snap-in coupler 80 has a first coupler member 90 and a second coupler member 70. The second coupler member 70 of the snap-in coupler 80 is similar to the second coupler

70 of the rivet coupler 50 shown in Figures 4 and 5 and the similar components are denoted with the same reference numbers.

The first coupler member 90 of the snap-in coupler 80 has a stem 92 and a recess 94. The recess 94 has a mouth 94a defining an opening around the stem 92.

5 The stem 92 has a connection end 92a where it is connected to the bottom 94a of the recess 94, similar to the first coupler 60 shown in Figure 4.

10 In this embodiment, however, the stem 92 has an enlarged head 92a. The enlarged head 92a has an enlarged section 92c that has a larger diameter d5 than the rest of the stem 92. This diameter d5 is larger than the diameter d3 of the tubular section 72b of the second coupler member 70. Alternatively, the enlarged head 92a may have bumps around its peripheral to provide a diameter d5 at the bumps.

15 In this embodiment, as shown in Figure 6a, the enlarged head 92b is slanted from the free end of diameter d6 to the enlarged section 92c of diameter d5. The diameter d6 is smaller than the diameter d5 to facilitate the insertion of the stem 92 into the tubular section 72b of the second coupler member 70. The diameter d5 is typically about 0.12 to 0.13 inches (about 3.05 to 3.30 mm) and the diameter d6 is typically about 0.095 to 0.105 inches (about 2.41 to 2.67 mm). The open angle α of the slant is typically about 60 to 80 degrees. The enlarged section 92c is typically formed at about 0.02 to 0.03 inches (about 0.51 to 0.76 mm) from the free end of the head 92b. Similar to the stem 62 shown in Figure 4, the height of the stem 92 typically about 0.07 to 0.09 inches (about 2.24 to 3.98 mm). However, these dimensions may vary depending on the design and purpose of the coupler 80.

25 When the friction assembly 20 is assembled, the noise damping shim 28 is brought into contact with the backing plate 26. During this contact, the enlarged head 92a of the stem 92 is snapped into the rim 72 at the connection section 72a through the tubular section 72b. As the enlarged head 92a is larger than the diameter of the tubular section 72b, the engagement between the first coupler element 60 and the second coupler element 70 is secured relative to torques during the brake operations. It is preferable that the length of the stem 92 is selected such that the enlarged head 92a snaps into opening defined by the connection section 72a

30

of the rim 72 only when the first and second coupler members 90 and 70 mate. In this case, the noise damping shim 28 is secured to the backing plate 26 not only in the direction of the torques but also in the direction perpendicular to the torques.

According to this embodiment, the noise damping shim 28 can be efficiently
5 assembled with the backing plate 26 in a single process without a need for extra components or complicated tools.

In the above embodiments, the stem of the coupler is formed on the backing plate and the stem receiving hole is formed on the shim. In a different embodiment, a stem may be formed on the shim and a stem receiving hole may be formed on the
10 backing plate. Also, in the above embodiments the components of the coupler generally have cylindrical shapes. In different embodiments, any different shapes of coupler components suitable for coupling the shim on the backing plate may be used. A single friction assembly may have several different types of couplers.

It is to be understood that what has been described is embodiments to the
15 invention. If the invention nonetheless is susceptible to certain changes and alternative embodiments fully comprehended by the spirit of the invention as described above, and the scope of the claims set out below.